

# Physics Handout 4

Advanced Mechanics & Oscillatory Systems  
Theoretical Analysis of Friction, Geometry, and Parametric Driving

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## Abstract

This document synthesizes the theoretical concepts underlying a set of advanced mechanics problems (Problems 01–06). The topics covered include friction-driven harmonic motion, geometric non-linearities in restoring forces, the tautochrone problem (Huygens pendulum), Coulomb damping in coupled systems, and the adiabatic invariants of parametric oscillators (variable length pendulums).

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# 1 Friction-Induced Oscillations (Problem 01)

## 1.1 Mechanism of Restoring Force

In standard Simple Harmonic Motion (SHM), the restoring force is typically elastic ( $F = -kx$ ). However, in Problem 01 (rod on rotating rollers), the restoring force arises from the **position-dependent normal force** modulating the kinetic friction.

Consider a rod of mass  $M$  on two rollers separated by distance  $a$ , rotating in opposite directions. When the rod is displaced by  $x$  from the center:

- **Torque Balance:** To prevent rotation, the normal forces  $N_1$  and  $N_2$  must adjust. Taking torques about the center of mass (CM):

$$N_1 \left( \frac{a}{2} + x \right) = N_2 \left( \frac{a}{2} - x \right)$$

Combined with  $N_1 + N_2 = Mg$ , this yields:

$$N_1 = \frac{Mg}{a} \left( \frac{a}{2} - x \right), \quad N_2 = \frac{Mg}{a} \left( \frac{a}{2} + x \right)$$

- **Friction Differential:** The net force is the difference in kinetic friction  $f_k = \mu N$ .

$$F_{net} = \mu N_1 - \mu N_2 = -\frac{2\mu Mg}{a} x$$

## 1.2 Conclusion

Because  $F_{net} \propto -x$ , the system undergoes SHM despite friction usually being a dissipative force. This is a classic example of **feedback** where the system's geometry converts a constant energy source (rotating rollers) into a conservative-like restoring force.

# 2 Geometric Linearization & Spring Constants (Problem 02)

## 2.1 Longitudinal vs. Transverse Stiffness

This problem highlights how the effective spring constant  $k_{eff}$  depends on the geometry of displacement.

1. **Longitudinal (Standard):** Displacement along the spring axis directly stretches the bonds.

$$F_{long} = -kx \implies \omega_{long} = \sqrt{k/m}$$

2. **Transverse (Geometric):** Displacement  $y$  perpendicular to the spring axis creates a restoring force via tension  $T$ .

$$F_{trans} = -2T \sin \theta \approx -2T \frac{y}{L}$$

Here, tension  $T = k(L - L_0)$ . If the spring is pre-stretched significantly ( $L \gg L_0$ ),  $T$  is roughly constant for small  $y$ , and the motion is linear.

$$k_{trans} = \frac{2T}{L} = \frac{2k(L - L_0)}{L}$$

**Key Insight**

Transverse stiffness is governed by *tension*, whereas longitudinal stiffness is governed by *elasticity*. This is analogous to the difference between sound waves in a solid (elasticity) and waves on a guitar string (tension).

### 3 Isochronism and the Cycloid (Problem 03)

#### 3.1 The Period of a Simple Pendulum

A simple pendulum is not strictly isochronous. The period  $T$  depends on amplitude  $\theta_0$ :

$$T(\theta_0) \approx 2\pi\sqrt{\frac{l}{g}} \left( 1 + \frac{1}{16}\theta_0^2 + \dots \right)$$

This non-linearity arises because the restoring force  $F \propto \sin \theta$  is "softer" than the required linear  $F \propto \theta$  at large angles.

#### 3.2 The Huygens (Cycloidal) Pendulum

To achieve perfect isochronism ( $T$  independent of amplitude), the particle must travel along a **tautochrone curve**.

- **Geometry:** The tautochrone is a cycloid.
- **Evolute of a Cycloid:** Huygens utilized the property that the evolute of a cycloid is another cycloid. By wrapping the pendulum string around cycloidal "cheeks," the bob traces a cycloid.
- **Physics:** On a cycloid, the effective arc length  $s$  relates to the restoring force such that  $F \propto -s$  exactly, not approximately. This ensures  $\omega$  is constant for any amplitude.

## 4 Coupled Oscillators & Coulomb Damping (Problem 04)

#### 4.1 Static vs. Kinetic Friction Interface

This problem introduces a **hybrid dynamical system** that switches regimes based on acceleration.

- **Sticking Regime** ( $a < a_{crit}$ ): The blocks move as a rigid body. The friction  $f_s$  acts as a constraint force, transmitting acceleration.
- **Slipping Regime** ( $a > a_{crit}$ ): The blocks decouple. Friction becomes a constant external force  $f_k = \mu N$  opposing motion relative to the slab.

#### 4.2 Energy Decay via Dry Friction

Unlike viscous damping ( $F \propto -v$ ), which decays amplitude exponentially ( $A \propto e^{-\gamma t}$ ), dry friction (Coulomb damping) removes a fixed amount of energy per cycle:

$$\Delta E_{cycle} = -4f_k A$$

Since  $E \propto A^2$ , we have  $\Delta(A^2) \propto A\Delta A$ . This leads to a **linear decay** of amplitude:

$$A(t) = A_0 - \alpha t$$

The oscillation stops completely in finite time, unlike viscous damping which asymptotically approaches zero.

## 5 Parametric Resonance & Adiabatic Invariants (Problem 05)

### 5.1 Variable Parameters in Oscillators

Consider an oscillator where the natural frequency  $\omega(t)$  changes slowly with time (e.g., changing pendulum length  $l(t)$ ). The equation of motion becomes:

$$\ddot{\theta} + \omega^2(t)\theta = 0$$

If  $\omega(t)$  varies slowly (adiabatic limit,  $\dot{\omega}/\omega \ll \omega$ ), the exact solution is difficult, but the energy evolves predictably.

### 5.2 Adiabatic Invariant

For a harmonic oscillator with slowly varying parameter  $\lambda$ :

$$I = \oint p dq = \frac{E}{\omega} = \text{constant}$$

This is the **Adiabatic Invariant**.

- For a pendulum,  $\omega = \sqrt{g/l}$ .
- If length  $l$  decreases (shortens),  $\omega$  increases.
- To keep  $E/\omega$  constant, Energy  $E$  must increase.

**Physical Interpretation:** Pulling the string up requires work against the centripetal component of tension. This work pumps energy into the system.

$$E(l) \propto \sqrt{g/l} \implies E \propto l^{-1/2}$$

Amplitude follows  $E \propto m\omega^2 A^2$ , yielding  $A \propto l^{-3/4}$ .

### 5.3 Parametric Resonance (The Swing Effect)

If the parameter  $l(t)$  is modulated periodically, specifically at  $\Omega = 2\omega_0$ , resonance occurs.

$$l(t) = l_0(1 + \epsilon \cos(2\omega_0 t))$$

This leads to the **Mathieu Equation**.

- **Mechanism:** If you shorten the pendulum when it is at the bottom (maximum tension/velocity) and lengthen it at the extremes (minimum tension), you do net positive work over a cycle.
- **Result:** The amplitude grows exponentially with time:

$$\theta(t) \sim e^{st} \cos(\omega t)$$

This is distinct from driven resonance (where  $\Omega = \omega_0$ ) which grows linearly.